

# Today's Outline

Lecture 2 Aug, 29th, 2011

I. Oscillations around stable P1/6

I. Last Lecture

II. ~~Wrap up~~ <sup>Recall</sup> Standard guess  
(2D Oscillations moved to section)

III Damped Oscillations

IV Damped oscillation Regions

II. SHO

$$m \ddot{x} = -kx$$

$$\dot{x} = r e^{rt} \quad \ddot{x} = r^2 e^{rt}$$

so,

$$r^2 m e^{rt} = -k e^{rt}$$

$$\Rightarrow r = \pm i \sqrt{\frac{k}{m}} = \pm i \omega_0$$

Solution is a linear combination (superposition):

equilibria are ubiquitous.

$$k = U''(x_{eq})$$

and we began to investigate the standard guess

$$x(t) = e^{rt} \quad \begin{array}{l} \text{switch to} \\ \text{book's} \\ \text{notation} \\ \text{(I used } \lambda \text{).} \end{array}$$

$$x(t) = C_1 e^{i\omega t} + C_2 e^{-i\omega t}$$

Euler's exquisite Creation:

$$e^{i\pi} + 1 = 0$$

Generally,

$$e^{i\theta} = \cos\theta + i\sin\theta$$

Amazing secret:

$$e^{i\theta} = \cos\theta + i\sin\theta$$

Use this to derive all of trigonometry!

Example:

$$\begin{aligned} e^{i2\theta} &= \cos 2\theta + i \sin 2\theta \\ &= (e^{i\theta})^2 = (\cos \theta + i \sin \theta)^2 \\ &= \cos^2 \theta - \sin^2 \theta + 2i \cos \theta \sin \theta \end{aligned}$$

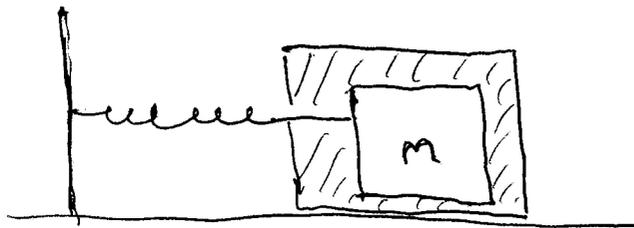
Real parts equal implies

$$\cos 2\theta = \cos^2 \theta - \sin^2 \theta = 2\cos^2 \theta - 1 = \dots$$

### III Damped <sup>Harmonic</sup> Oscillations ~~oscillations~~

Hooke's law:  $F_{\text{spring}} = -kx$

Immerse the mass in a tank of viscous medium.



Challenge 1: For the duration of this course never look up a trig formula. #2/6

Challenge 2: Figure out four distinct ways to write the general solution to

$$m\ddot{x} = -kx$$

using complex numbers (solution in your text).

Assume force is proportional to velocity and opposite in direction

$$f_{\text{viscous}} = -bv$$

(e.g. neglecting turbulence)

Newton's <sup>2<sup>nd</sup></sup> second law:

$$F_{\text{net}} = F_{\text{spring}} + f_{\text{viscous}} = ma = m\ddot{x}$$

So,

$$m\ddot{x} = -kx - b\dot{x}$$

or

$$m\ddot{x} + b\dot{x} + kx = 0$$

or

$$\ddot{x} + \frac{b}{m}\dot{x} + \omega_0^2 x = 0,$$

Recall  $\omega_0 = \sqrt{k/m}$ .

Introduce shorthand  $\beta = \frac{b}{2m}$ ,

$$\ddot{x} + 2\beta\dot{x} + \omega_0^2 x = 0$$

Call these

$$r_1 = -\beta + \sqrt{\beta^2 - \omega_0^2} \quad r_2 = -\beta - \sqrt{\beta^2 - \omega_0^2}$$

$$x(t) = C_1 e^{r_1 t} + C_2 e^{r_2 t}$$

Important differences in this solution depending on regime of the parameters  $\beta, \omega_0$ .

IV Regimes:

Three Regimes:

(i) Overdamped:  $\beta > \omega_0$ .

Solve for  $x(t)$ ? Use standard guess,  $x = e^{rt}$ . P3/6

$$\dot{x} = r e^{rt} \quad \ddot{x} = r^2 e^{rt}$$

$$r^2 e^{rt} + 2\beta r e^{rt} + \omega_0^2 e^{rt} = 0$$

$$\Rightarrow r^2 + 2\beta r + \omega_0^2 = 0$$

$$\Rightarrow r = \frac{-2\beta \pm \sqrt{4\beta^2 - 4\omega_0^2}}{2}$$

$$= -\beta \pm \sqrt{\beta^2 - \omega_0^2}$$

Then  $B = \sqrt{\beta^2 - \omega_0^2}$  is real and the general solution is

$$x(t) = C_1 e^{(\beta+B)t} + C_2 e^{(-\beta-B)t}$$

Both terms are declining exponentials (note:  $\beta > B > 0$ ).

[Aside: For exponentials the "characteristic time",  $\tau$ , is the time it takes to get to  $1/e$  of its original value]

In the overdamped case there are two characteristic times

$$\tau_1 = \frac{1}{-\beta + B} \quad ; \quad \tau_2 = \frac{1}{\beta + B}.$$

In fact, for Huge damping  $\beta \approx B$  and  $\tau_1 \rightarrow \infty$

$$= e^{-bt} (D_1 \cos \omega_1 t + D_2 \sin \omega_1 t)$$

with  $D_1 = C_1 + C_2$ ,  $D_2 = i(C_1 - C_2)$ .

$$x(t) = A e^{-\beta t} \cos(\omega_1 t - \delta)$$

Sinusoidal oscillations with exponentially decreasing amplitude  $A e^{-\beta t}$ .

(2) Underdamped:  $\beta < \omega_0$ . Then P4/6

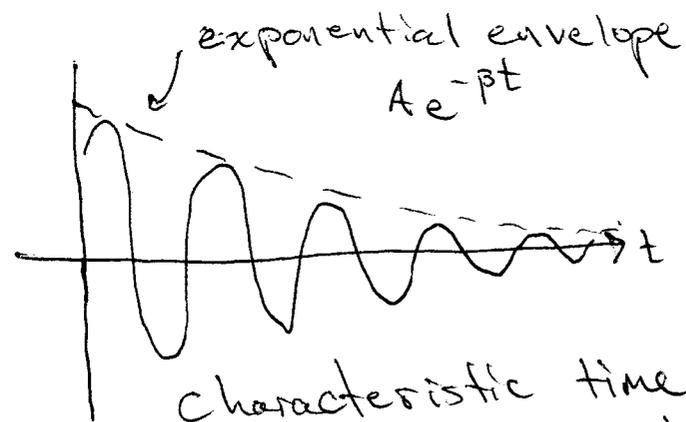
$$\omega_1 \equiv \sqrt{\omega_0^2 - \beta^2} \text{ is real}$$

$$\Gamma_{1,2} = -\beta \pm i\omega_1$$

and the general solution is

$$x(t) = C_1 e^{(-\beta + i\omega_1)t} + C_2 e^{(-\beta - i\omega_1)t}$$

$$= e^{-\beta t} (C_1 [\cos \omega_1 t + i \sin \omega_1 t] + C_2 [\cos \omega_1 t - i \sin \omega_1 t])$$



Two times at work here: damping  $\tau = 1/\beta$ ; and the period of the oscillations

$$T = \frac{2\pi}{\omega_1}$$

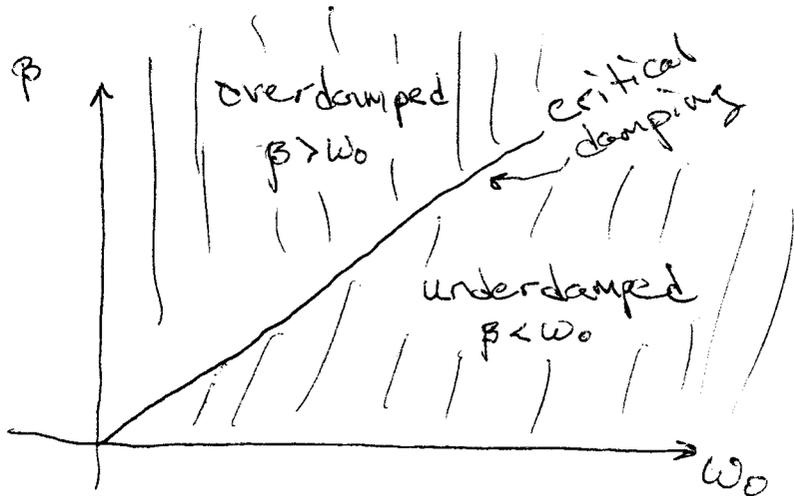
Question: How many oscillations in one characteristic time?

$$\# = \frac{\tau}{T} = \frac{1}{\beta} \cdot \frac{\omega_1}{2\pi} = \frac{1}{\pi} \left( \frac{\omega_1}{2\beta} \right)$$

$$\text{Quality factor} \equiv Q \equiv \omega_1 / 2\beta$$

"How many times does it ring?"

IF there's time:



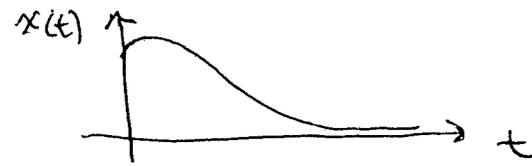
(3) Critical Damping: P5/6

$$\beta = \omega_0 \Rightarrow r_1 = r_2$$

General solution is now,

$$x(t) = C_1 e^{-\beta t} + C_2 t e^{-\beta t}$$
$$= e^{-\beta t} (C_1 + C_2 t)$$

again  $\tau = 1/\beta$



The derivatives in our damped oscillator EOM can be collected into a differential operator:

$$D \equiv \frac{d^2}{dt^2} + 2\beta \frac{d}{dt} + \omega_0^2$$

$$Dx = \ddot{x} + 2\beta \dot{x} + \omega_0^2 x$$

This operator is linear Superposition

$$D(ax) = aDx \quad D(x_1 + x_2) = Dx_1 + Dx_2$$

Linear operators with constant coefficients are the ones for which the standard guess works. Show it!