

Today's Outline

I Last Lecture
& finish double pendulum

Lecture 29

November 7th, 2011

I Last Lecture

P1/4

We studied the double pendulum and found the E.O.M

$$(m_1 + m_2) L_1^2 \ddot{\phi}_1 + m_2 L_1 L_2 \ddot{\phi}_2 = -(m_1 + m_2) g L_1 \phi_1$$

$$m_2 L_1 L_2 \ddot{\phi}_1 + m_2 L_2^2 \ddot{\phi}_2 = -m_2 g L_2 \phi_2$$

These E.O.M are equivalent to

$$\tilde{M} \ddot{\vec{\phi}} = -\tilde{K} \vec{\phi}$$

with $\vec{\phi} = (\phi_1, \phi_2)$ and

$$\tilde{M} = \begin{pmatrix} (m_1 + m_2) L_1^2 & m_2 L_1 L_2 \\ m_2 L_1 L_2 & m_2 L_2^2 \end{pmatrix}, \quad \tilde{K} = \begin{pmatrix} (m_1 + m_2) g L_1 & 0 \\ 0 & m_2 g L_2 \end{pmatrix}$$

In this form the problem is exactly like our toy model:

Guess: $\vec{\phi}(t) = \text{Re} \vec{Z}(t)$, $\vec{Z}(t) = \vec{a} e^{i\omega t} = \begin{pmatrix} a_1 \\ a_2 \end{pmatrix} e^{i\omega t}$

and

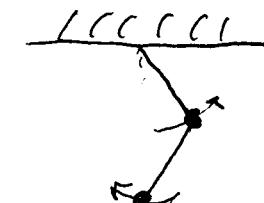
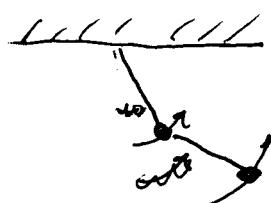
Solve: $\begin{cases} \det(\tilde{K} - \omega^2 \tilde{M}) = 0 & \text{for normal freq's} \\ (\tilde{K} - \omega^2 \tilde{M}) \vec{a} = 0 & \text{for normal modes} \end{cases}$

For example when $m_1 = m_2 = m$ and $l_1 = l_2 = l$ and we let $\omega_0 = \sqrt{g/l}$ then

$$\omega_1 = \sqrt{2 - \sqrt{2}} \quad \omega_0 \approx 0.77 \omega_0$$

$$\omega_2 = \sqrt{2 + \sqrt{2}} \quad \omega_0 \approx 1.85 \omega_0$$

$$\vec{a}_1 = \begin{pmatrix} 1 \\ \sqrt{2} \end{pmatrix} A_1 e^{-i\omega_1 t}, \quad \vec{a}_2 = \begin{pmatrix} 1 \\ -\sqrt{2} \end{pmatrix} A_2 e^{-i\omega_2 t}$$



II Lagrangian Approach: the general case

A system of n degrees of freedom oscillating about a point of stable equilibrium. We will describe the system (assumed holonomic) by n generalized coordinates: q_1, \dots, q_n or $\vec{q} = (q_1, \dots, q_n)$ (e.g. $\vec{q} = (x_1, x_2)$ for toy model or $\vec{q} = (\phi_1, \phi_2)$ for double pendulum).

Then

$$\begin{aligned} T &= \frac{1}{2} \sum_{\alpha} m_{\alpha} \dot{\vec{r}}_{\alpha} \cdot \dot{\vec{r}}_{\alpha} \\ &= \frac{1}{2} \sum_{\alpha} m_{\alpha} \left(\sum_j \frac{\partial \vec{r}_{\alpha}}{\partial q_j} \dot{q}_j \right) \cdot \left(\sum_k \frac{\partial \vec{r}_{\alpha}}{\partial q_k} \dot{q}_k \right) \\ &= \frac{1}{2} \sum_{i,j,k} A_{ijk} \dot{q}_i \dot{q}_j \dot{q}_k \end{aligned}$$

with $A_{ijk} \equiv \sum_{\alpha} m_{\alpha} \left(\frac{\partial \vec{r}_{\alpha}}{\partial q_i} \right) \left(\frac{\partial \vec{r}_{\alpha}}{\partial q_k} \right)$ and

$$A_{ijk} = A_{ijk}(q_1, \dots, q_n) = A_{ijk}(\vec{q})$$

As always the K.E. is

$$T = \frac{1}{2} \sum_{\alpha} \frac{1}{2} m_{\alpha} \dot{\vec{r}}_{\alpha}^2 \quad \alpha = (1, \dots, N) \quad \# \text{ of particles}$$

We write this in terms of the generalized coordinates by using the transformation

$$\vec{r}_{\alpha} = \vec{r}_{\alpha}(q_1, \dots, q_n)$$

By the chain rule

$$\dot{\vec{r}}_{\alpha} = \sum_{i=1}^n \frac{\partial \vec{r}_{\alpha}}{\partial q_i} \dot{q}_i$$

We assume the forces are conservative, so that

$$U = U(q_1, \dots, q_n) = U(\vec{q})$$

Next, we assumed \vec{q}_0 is a stable equilibrium and by translating our origin take $\vec{q}_0 = \vec{0}$. For small oscillations we can Taylor expand

$$U(\vec{q}) = U(0) + \sum_j \frac{\partial U}{\partial q_j} \vec{q}_j + \frac{1}{2} \sum_{i,j,k} \frac{\partial^2 U}{\partial q_i \partial q_j \partial q_k} \vec{q}_i \vec{q}_j \vec{q}_k$$

constant which can be dropped. $+ \dots$

If we let $K_{jk} = \left. \frac{\partial^2 U}{\partial \dot{q}_j \partial \dot{q}_k} \right|_{\ddot{q}=0}$ (note that $K_{jk} = K_{kj}$) and neglect higher order terms, we have,

$$U = U(\ddot{q}) = \frac{1}{2} \sum_{j,k} K_{jk} \dot{q}_j \dot{q}_k$$

Actually since we're dropping higher order terms

already of 2nd order

$$T = \frac{1}{2} \sum_{j,k} A_{jk}(\ddot{q}) \dot{q}_j \dot{q}_k \approx \frac{1}{2} \sum_{j,k} M_{jk} \dot{q}_j \dot{q}_k$$

get solvable linear E.O.M out of 'em.

Let's do it:

$$\mathcal{L} = T - U$$

$$\begin{aligned} \frac{\partial \mathcal{L}}{\partial \dot{q}_i} &= \frac{\partial}{\partial \dot{q}_i} \left(\frac{1}{2} \sum_{j,k} M_{jk} \dot{q}_j \dot{q}_k \right) - \frac{\partial U}{\partial \dot{q}_i} \\ &= \frac{1}{2} \left(\sum_{j,k} M_{jk} \dot{q}_j \dot{q}_k + \sum_{j,k} M_{jk} \dot{q}_j \underset{\cancel{\dot{q}_k}}{\dot{q}_k} \right) \\ &= \frac{1}{2} \left(\sum_k M_{ik} \dot{q}_k + \sum_j M_{ji} \dot{q}_i \right) = \sum_j M_{ij} \dot{q}_i \end{aligned}$$

\uparrow symmetric

where

$M_{jk} = A_{jk}(\ddot{q})$
is a matrix of constants.

Note that this simplifies T so that $T = T(\ddot{q}, \dot{q})$ reduces to $T = T(\dot{q})$.

What have we achieved? Once again T and U are homogeneous quadratic functions of the \dot{q} 's and q 's respectively! Again we'll

$$\frac{d}{dt} \left(\frac{\partial \mathcal{L}}{\partial \dot{q}_i} \right) = \sum_j M_{ij} \ddot{q}_j$$

$$\begin{aligned} \frac{\partial \mathcal{L}}{\partial q_i} &= \frac{\partial T}{\partial q_i} - \frac{\partial U}{\partial q_i} \\ &= -\frac{1}{2} \frac{\partial}{\partial q_i} \left(\sum_{j,k} K_{jk} \dot{q}_j \dot{q}_k \right) \\ &= -\sum_j K_{ij} \dot{q}_j \end{aligned}$$

$\Rightarrow \hat{M} \ddot{\vec{q}} = -\hat{K} \vec{q}$

Again we guess,

$$\vec{g}(t) = \text{Re } \vec{z}(t) \quad \text{with } \vec{z}(t) = \vec{a} e^{i\omega t}$$

and

solve: $\begin{cases} \det(\tilde{K} - \omega^2 \tilde{M}) = 0 & \text{normal freqs} \\ (\tilde{K} - \omega^2 \tilde{M}) \vec{a} = 0 & \text{normal modes} \end{cases}$

The det equation is an n th degree polynomial in ω^2 which we solve for the n normal frequencies. Then we determine the n \vec{a} vectors. And

Your text goes through another example in detail, take a look at it.

We've completed the core of the course:

Oscillations

Calculus of variations

Lagrangian Mechanics

Central Forces

Noninertial Frames

Rigid Bodies

Normal modes

Special Topics:
(to come)

Nonlinear Mechanics
& chaos

Hamiltonian Mechanics

finally the general solution P4/4 is a linear combination of the n normal modes.

Notice that in fact you needn't write down the Lagrangian at all for these problems anymore; you can just find \tilde{M} and \tilde{K} from the general formulae we wrote down.

Continuum Mechanics