

Mechanics

P1/5

Day 2 I. Dot notation:

$$\dot{x} = \frac{dx}{dt} \quad \text{and} \quad \ddot{x} = \frac{d^2x}{dt^2}$$

Oscillations are ubiquitous because the PE is generically quadratic in the vicinity of a stable equilibrium



Here we can't Taylor expand because $U(y)$ is not differentiable at $y=0$.

II. We will change perspective in this course

$$F = ma = -Kx$$

$$\downarrow \quad m \frac{d^2x}{dt^2} = -Kx$$

\downarrow $m \ddot{x} = -Kx$ viewed as a differential eqn.

We will learn techniques for writing

Today

- I Last time
- II Standard guess
- III Damped Oscillations
- IV Regimes

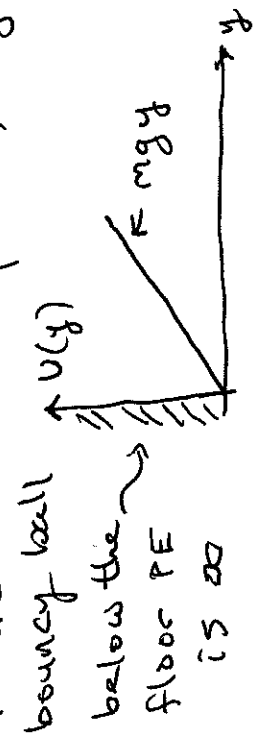
Second derivative test:

$$\text{stable: } \frac{d^2U}{dx^2} > 0 \quad \text{unstable: } \frac{d^2U}{dx^2} < 0$$

This means we can find the effective spring constant using

$$K = U''(x_{eq})$$

There are exceptions, e.g. the



down eqns of motion (EOM), but also for solving them.

Terminology:

Ordinary diff. eqn. (ODE): only ordinary derivatives

Partial diff. eqn. (PDE): partial derivatives

Order of diff. eqn.: the highest derivative that appears.

Take the simple harmonic oscillator

and we've reduced the ODE to an algebraic eqn:

$$\Rightarrow r = \pm i \sqrt{\frac{k}{m}} \equiv \pm i \omega_0$$

General solution is a linear

combo (superposition):

$$x(t) = C_1 e^{i\omega_0 t} + C_2 e^{-i\omega_0 t}$$

constants ↗

A 2nd order ODE always has a

general solution depending on 2 const's.

as an example, it's a 2nd order ODE. The most important method of solution is the "standard guess":

$$x(t) = e^{rt}$$

The EOM is

$$m \ddot{x} = -kx$$

With this guess we have

$$\dot{x} = r e^{rt}, \quad \ddot{x} = r^2 e^{rt}$$

$$\text{SO, } m r^2 e^{rt} = -k e^{rt}$$

Physically, it corresponds to the fact that you need boundary conditions:

$$x(0) \equiv x_0 = C_1 + C_2 \quad \text{and}$$

$$\dot{x}(0) \equiv v_0 = i\omega_0 C_1 - i\omega_0 C_2.$$

Euler's exquisite creation:

$$e^{i\pi} + 1 = 0$$

Generally,

$$e^{i\theta} = \cos\theta + i\sin\theta$$

Amazing secret: $e^{i0} = \cos 0 + i\sin 0$.

Challenge 1: For the duration of this course never look up a trig. formula.

Challenge 2: Figure out four distinct ways to write the general solution to

$m \ddot{x} = -kx$
using complex numbers
(solution in your text)

Assume force is proportional to velocity and opposite in direction

$F_{\text{viscous}} = -b\dot{x}$

(e.g. neglecting turbulence)

Newton's 2nd law:

$F_{\text{net}} = F_{\text{spring}} + F_{\text{viscous}} = m\ddot{x} = m\ddot{x}$

Can use this to derive all of trigonometry!

Example: $e^{iz\theta} = \cos\theta + i\sin\theta$

$= (\cos\theta + i\sin\theta)^2$

$= \cos^2\theta - \sin^2\theta + 2i\cos\theta\sin\theta$

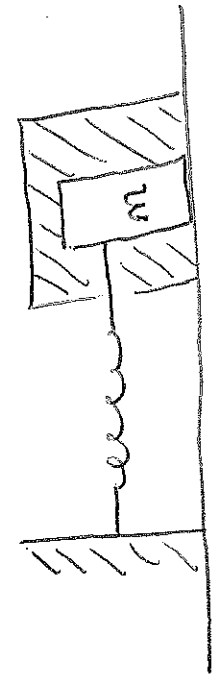
Real parts equal implies

$\cos 2\theta = \cos^2\theta - \sin^2\theta = 2\cos^2\theta - 1$ etc.

III Damped Harmonic Oscillations

Hooke's Law: $F_{\text{spring}} = -kx$

Immerse the mass in a tank of viscous medium.



So, $m\ddot{x} = -kx - b\dot{x}$

or $m\ddot{x} + b\dot{x} + kx = 0$

or $\ddot{x} + \frac{b}{m}\dot{x} + \omega_0^2 x = 0$

Recall, $\omega_0 = \sqrt{k/m}$ and
introduce shorthand $\beta = \frac{b}{2m}$
so that

$$\ddot{x} + 2\beta\dot{x} + \omega_0^2 x = 0$$

Call these

$$r_1 = -\beta + \sqrt{\beta^2 - \omega_0^2}, \quad r_2 = -\beta - \sqrt{\beta^2 - \omega_0^2}$$

$$x(t) = C_1 e^{r_1 t} + C_2 e^{r_2 t}$$

Important differences in this
solution depending on regime
of the parameters β, ω_0 .

IV Three Regimes

(1) Over-damped: $\beta > \omega_0$.

Solve for $x(t)$? Use Standard P4/5
guess, $x = e^{rt}$

$$\dot{x} = r e^{rt}, \quad \ddot{x} = r^2 e^{rt}$$

$$r^2 e^{rt} + 2\beta r e^{rt} + \omega_0^2 e^{rt} = 0$$

$$\Rightarrow r^2 + 2\beta r + \omega_0^2 = 0$$

$$\Rightarrow r = \frac{-2\beta \pm \sqrt{4\beta^2 - 4\omega_0^2}}{2}$$

$$= -\beta \pm \sqrt{\beta^2 - \omega_0^2}$$

Then $\beta \equiv \sqrt{\beta^2 - \omega_0^2}$ is real
and the general solution is

$$x(t) = C_1 e^{(-\beta + \beta)t} + C_2 e^{(-\beta - \beta)t}$$

Both terms are damping
exponentials (Note $\beta > \beta > 0$).

[Aside: For exp's the "characteristic
time", τ , is the time it
takes to get to $1/e$ of its
original value.]

(2) Underdamped: $\beta < \omega_0$. P5/5

Then $\omega_1 \equiv \sqrt{\omega_0^2 - \beta^2}$ is real

$$r_{1,2} = -\beta \pm i\omega_1$$

and the general soln. is

$$x(t) = C_1 e^{(-\beta + i\omega_1)t} + C_2 e^{(-\beta - i\omega_1)t}$$

$$= e^{-\beta t} (C_1 [\cos \omega_1 t + i \sin \omega_1 t] + C_2 [\cos \omega_1 t - i \sin \omega_1 t])$$

In the overdamped regime there are two characteristic times

$$\tau_1 = \frac{1}{\beta - B} ; \tau_2 = \frac{1}{\beta + B}$$

In fact, for huge damping

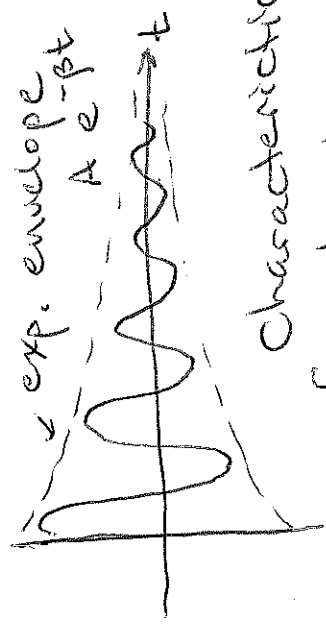
$$\beta \approx B \text{ and } \tau_1 \rightarrow \infty$$

$$= e^{-\beta t} (D_1 \cos \omega_1 t + D_2 \sin \omega_1 t)$$

with $D_1 = C_1 + C_2, D_2 = i(C_2 - C_1)$

$$x(t) = A e^{-\beta t} \cos(\omega_1 t - \delta)$$

Sinusoidal oscillations w/ exp. decreasing amplitude $A e^{-\beta t}$



exp. envelope
 $A e^{-\beta t}$

Characteristic time for damping $\tau = 1/\beta$

Two times at work here: damping $\tau = 1/\beta$ and the period of the oscillations

$$T = \frac{2\pi}{\omega_1}$$